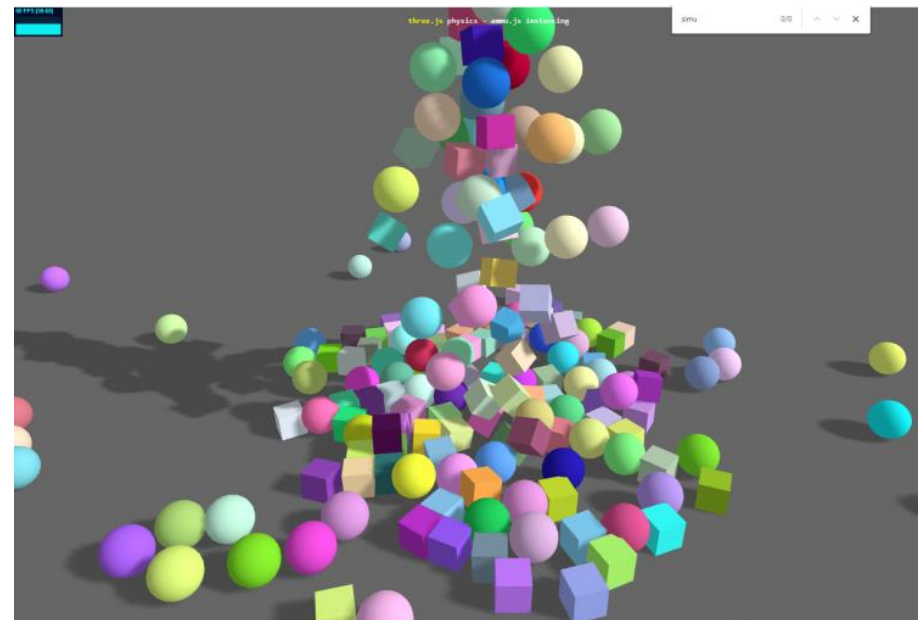


CG3 – Rigid Body Simulation

Rigid Body Simulation



<https://github.com/chandlerprall/Physijs>



https://threejs.org/examples/#physics_ammo_instancing



- ◆ Throw box into rotational motions along main axes
- ◆ One axis results in unstable rotations, why?

- ◆ Check out:
 - ◆ Intermediate axis theorem
 - ◆ Tennis Racket Theorem
 - ◆ Dzhanibekov Effect, or

<https://www.camein.com/rotating-bodies-dzhanibekov-effect/>

◆ Motivation

◆ Placement $\underline{\mathbf{X}}; \mathbf{R}$

◆ Kinematics $\vec{\mathbf{V}}; \vec{\omega}$

◆ Dynamics $M, \vec{\mathbf{P}}, \vec{\mathbf{F}}; \mathbf{I}, \vec{\mathbf{L}}, \vec{\mathbf{T}}$

◆ Equations of Motion

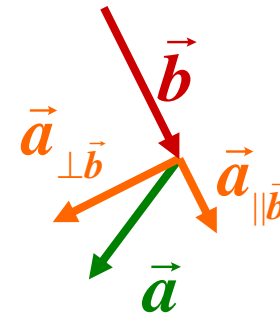
$$\vec{\mathbf{y}} = \begin{pmatrix} \underline{\mathbf{X}} \\ \mathbf{R} \\ \vec{\mathbf{P}} \\ \vec{\mathbf{L}} \end{pmatrix} \dots \dot{\vec{\mathbf{y}}} = \begin{pmatrix} M^{-1} \vec{\mathbf{P}} \\ \mathbf{I}^{-1} \vec{\mathbf{L}} \\ \vec{\mathbf{F}} \\ \vec{\mathbf{T}} \end{pmatrix}$$

Literatur

- ◆ Baraff, David. Rigid Body Simulation I + II (Siggraph course, Physically Based Modeling 1997) [link](#)
- ◆ Nolting, Wolfgang, Grundkurs Theoretische Physik 1, Klassische Mechanik, Band 1 Springer, 2. Auflage 2003

Vectors – Notation

- ◆ In the following, force **vectors** are usually decomposed into components that are **parallel** or **perpendicular** to another vector
- ◆ For this we introduce the two vectorial **short notation** that project perpendicularly or onto the vector.
- ◆ The same **notation** is used for the **lengths** of the respective components, except that the symbol is not written bold and without a vector.



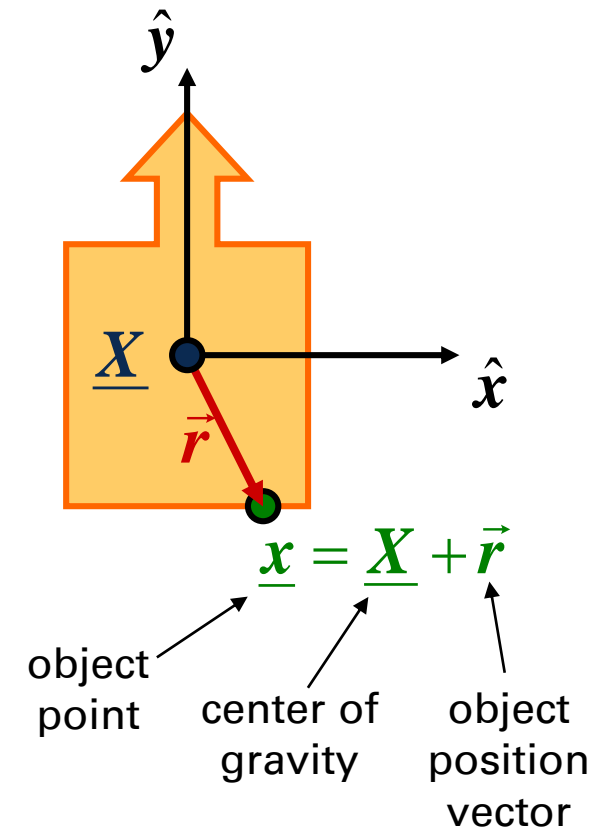
$$\vec{a}_{\parallel \vec{b}} = (\vec{a}^T \vec{b}) \vec{b} / \vec{b}^2$$

$$\vec{a}_{\perp \vec{b}} = \vec{a} - \vec{a}_{\parallel \vec{b}}$$

$$a_{\parallel \vec{b}} = \left\| \vec{a}_{\parallel \vec{b}} \right\|$$

$$a_{\perp \vec{b}} = \left\| \vec{a}_{\perp \vec{b}} \right\|$$

- ◆ A rigid body can be positioned in space by means of an **Euclidean transformation**, i.e. a rotation and a translation.
- ◆ For this one defines a local **object coordinate system O** per rigid body.
- ◆ The **natural origin** is the **center of mass \underline{X}** of the rigid body.
- ◆ A **natural orientation** results from the **inertia tensor** (see slides [12ff](#)).
- ◆ In 2D the **orientation** is defined by an **angle α** . In general, a rotation **matrix R** can be used.



Discrete Case

- Rigid body is decomposed into discrete point masses \underline{x}_i, m_i .
- The center of mass \underline{X} is the average position weighted with the point mass, which is derived from the affine combination

$$\underline{X} = \frac{1}{M} \sum_i m_i \underline{x}_i$$

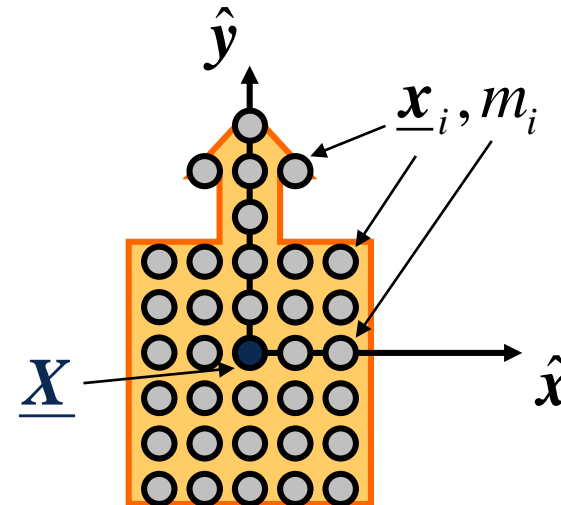
- with the total mass M :

$$M = \sum_i m_i$$

Continuous Case

- here one defines the mass density

$$\rho = dM/dV$$



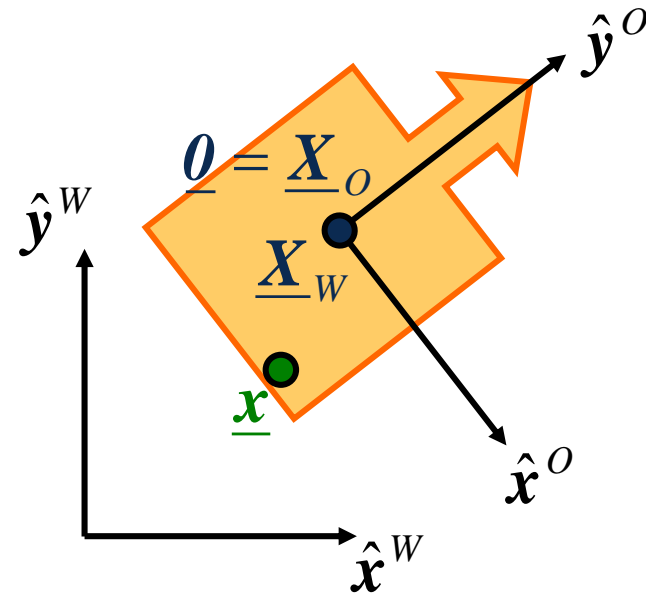
- summations become integrals:

$$M = \int_V \rho(\underline{x}) dV$$

$$\underline{X} = \frac{1}{M} \int_V \rho(\underline{x}) \underline{x} dV$$

Notation

- W and O stand for **world** and **object coordinate** system.
- A **subscript** defines the coordinate system in which the vector components are given.
- For base vectors, the additional **superscripts** indicate which coordinate system is spanned by the base.
- For positioning, an **Euclidean transformation** from the natural object coordinates into world coordinates is given.



$$\underline{x}_W = \mathbf{R}(\underline{x}_O - \underline{X}_O) + \underline{X}_W$$

$$\vec{r}_W = \mathbf{R}\vec{r}_O$$

$$\mathbf{R} = \begin{pmatrix} \hat{x}^O & \hat{y}^O \\ \hat{x}^W & \hat{y}^W \end{pmatrix}$$

$$\underline{x}_O = \mathbf{R}^T(\underline{x}_W - \underline{X}_W) + \underline{X}_O$$

$$\vec{r}_O = \mathbf{R}^T\vec{r}_W$$

Kinematics

- By derivation with respect to time one receives two contributions to the velocity (note that \vec{r}_O does not change over time)

$$\underline{x}_W = \mathbf{R}\vec{r}_O + \underline{X}_W$$

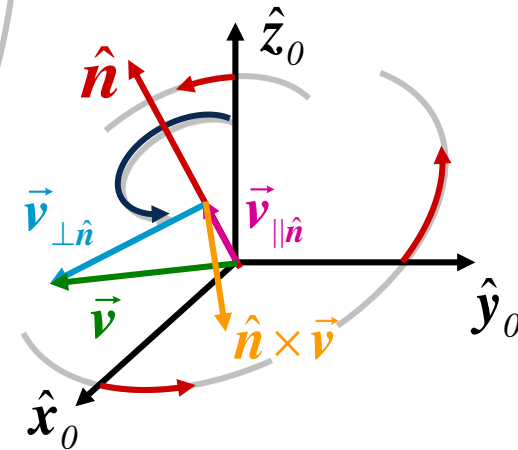
$$\underline{\dot{x}}_W = \dot{\mathbf{R}}\vec{r}_O + \underline{\dot{X}}_W$$

$$\underbrace{\begin{pmatrix} \omega_x \\ \omega_y \\ \omega_z \end{pmatrix}}_{\vec{\omega}} \rightarrow \underbrace{\begin{pmatrix} 0 & -\omega_z & \omega_y \\ \omega_z & 0 & -\omega_x \\ -\omega_y & \omega_x & 0 \end{pmatrix}}_{\vec{\omega}^*}$$

Linear velocity

- Describes the uniform motion of the rigid body (which can also rotate around the center of mass).

$$\underline{V}_W = \underline{\dot{X}}_W$$



Angular velocity

- Describes the **change of orientation**
- For infinitesimal small dt rotation can be assumed to be **constant around** fixed **axis \hat{n}**
- with **angular velocity** defined as

$$\vec{\omega} = \frac{d\alpha}{dt} \hat{n}$$

$$\dot{\mathbf{R}} = \underbrace{(\mathcal{R}(\hat{n}, d\alpha)\mathbf{R} - \mathbf{R})}_{\vec{\omega}^* \mathbf{R}} / dt$$

$$\hat{n}\hat{n}^T + \underbrace{(\mathbf{1} - \hat{n}\hat{n}^T)}_1 \underbrace{\cos d\alpha}_1 + \hat{n}^* \underbrace{\sin d\alpha}_{d\alpha}$$

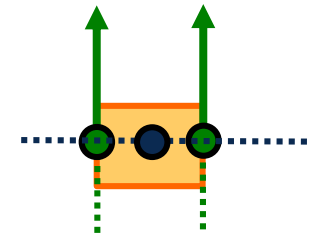
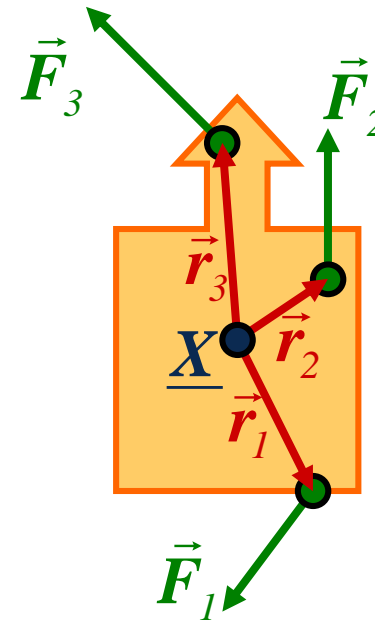
$$\stackrel{d\alpha \rightarrow 0}{\Rightarrow} \mathbf{1} + \hat{n}^* d\alpha$$

$$\dot{\mathbf{R}} = \vec{\omega}^* \mathbf{R} = \hat{n}^* \frac{d\alpha}{dt} \mathbf{R} \quad \text{mit } \vec{\omega} = \frac{d\alpha}{dt} \hat{n}$$

- In the following we assume the **world coordinate system as default** even if the subscript W is not given.
- As with kinematics, dynamics can be split into **linear and angular motion**.

Linear Dynamics

- All forces acting on the body are applied to the center of mass and added to the total force which, according to Newton, changes the linear velocity and the linear momentum.
- The procedure is equivalent to the one for point masses



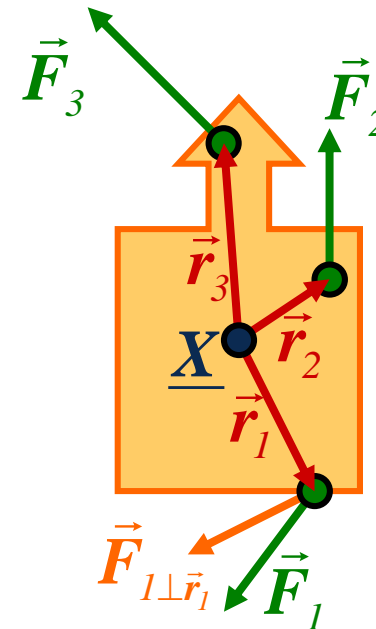
simple example that shows, why force needs to be transported to centroid also orthogonal to **force action lines**

$$\vec{F}_{\text{tot}} = \sum_i \vec{F}_i = M\dot{\vec{V}} = \dot{\vec{P}}$$

with linear momentum $\vec{P} = M\vec{V}$

Angular Dynamics

- Torque \vec{T} is the rotational equivalent to force
- Torque measures lever action with that a force \vec{F}_i acts on center of mass \underline{X} when force is applied to an object position vector \vec{r}_i .
- Torque points along the rotation axis and in 2D orthogonal to 2D plane (up or down)
- The absolute value T can be computed from the length of \vec{r}_i , which is the length of the lever, and the component of the force orthogonal to \vec{r}_i .
- Any force \vec{F}_i therefore acts twice – once for linear and once for the angular dynamics



only 2D

3D case

$$T_i = \pm r_i F_{i \perp \vec{r}_i}$$

$$\vec{T}_i = \vec{r}_i \times \vec{F}_i$$

$$T_{\text{tot}} = \sum_i T_i$$

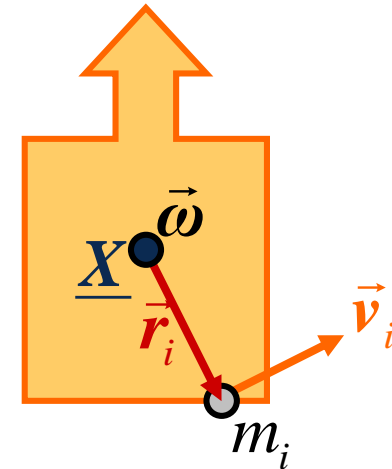
$$\vec{T}_{\text{tot}} = \sum_i \vec{T}_i$$

Rotation & Newton's Laws

- For now, we will look at only **one point mass**, which circles around the center of mass with the angular velocity.
- For an acceleration on the orbit a force is needed which accelerates the point mass according to **Newton's 2nd law**.
- This force can be directly converted into the corresponding **torque**.
- If one uses the vector identity

$$\vec{a} \times (\vec{b} \times \vec{c}) = (\vec{a} \cdot \vec{c})\vec{b} - (\vec{a} \cdot \vec{b})\vec{c}$$

the angular acceleration can be factored out also in the vectorial case.



only 2D

$$v_i = r_i \omega$$

$$F_i = m_i \dot{v}_i$$

$$= m_i r_i \dot{\omega}$$

$$T_i = m_i r_i^2 \dot{\omega}$$

3D case

$$\vec{v}_i = \vec{\omega} \times \vec{r}_i$$

$$\vec{F}_i = m_i \dot{\vec{v}}_i$$

$$= m_i \dot{\vec{\omega}} \times \vec{r}_i$$

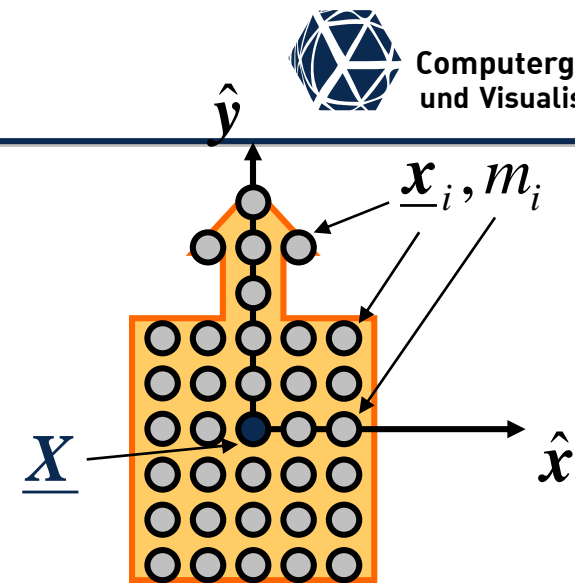
$$\vec{T}_i = m_i \vec{r}_i \times (\dot{\vec{\omega}} \times \vec{r}_i)$$

$$= m_i (\vec{r}_i^2 \mathbf{1} - \vec{r}_i \vec{r}_i^T) \dot{\vec{\omega}}$$

Rotation & Newton's Laws

- Summing the contributions of all point masses yields the total torque
- This is **proportional** to the **angular acceleration** in 2D.
- The proportionality constant is called the **moment of inertia** and is the equivalent to the mass in linear dynamics.
- It grows **quadratically in the distance** to the axis of rotation.
- In 3D we obtain a **symmetric 3x3-matrix**
- In the **continuous case**, the inertia tensor results from

$$\mathbf{I} = \int \rho(\vec{r}) (\vec{r}^2 - \vec{r}\vec{r}^T) d\vec{r}$$



2D case $T_{ges} = \left(\sum_i m_i r_i^2 \right) \dot{\omega} = I \dot{\omega}$
 moment of inertia: $I = \sum_i m_i r_i^2$

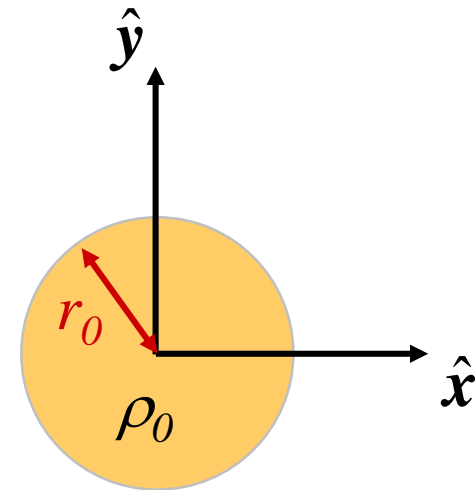
3D case $\vec{T}_{ges} = \underbrace{\left(\sum_i m_i (\vec{r}_i^2 - \vec{r}_i \vec{r}_i^T) \right)}_I \dot{\vec{\omega}}$

sym. inertia tensor: $I = \sum_i m_i (\vec{r}_i^2 - \vec{r}_i \vec{r}_i^T) \in \mathbf{R}^{3 \times 3}$

$$= \sum_i m_i \begin{pmatrix} y_i^2 + z_i^2 & -x_i y_i & -x_i z_i \\ -x_i y_i & x_i^2 + z_i^2 & -y_i z_i \\ -x_i z_i & -y_i z_i & x_i^2 + y_i^2 \end{pmatrix}$$

Example of circular disk of radius r_0

- ◆ This is the 2D case and therefore a **moment of inertia** is calculated.
- ◆ Radius is r_0 and density constant equals ρ_0 over total disk
- ◆ The best way is to transform the integral into **cylinder coordinates** and integrate them by angle (yields a factor of 2π) and radius.
- ◆ The result can be **interpreted** in such a way that the circular disk has the same inertia with regard to rotation **as a ring** or point mass **at radius $1/\sqrt{2} r_0$**

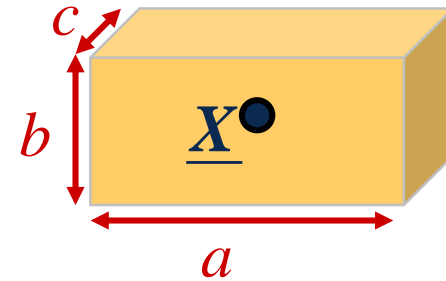


$$I = \int \rho(\vec{r}) r^2 dy dx$$

$$\begin{aligned} I &= \int_0^{r_0} \int_0^{2\pi} \rho(\vec{r}) r^2 [r \cdot d\phi \cdot dr] \\ &= 2\pi \int_0^{r_0} \rho_0 r^3 dr = \frac{1}{2} \pi \rho_0 r_0^4 \\ &= \frac{1}{2} M_0 r_0^2 \end{aligned}$$

Example Cuboid

- It is important to place the **origin** in the **center of mass**.
- By **alignment** to the **main axes**, a **diagonal matrix** with three main moments of inertia is obtained I_x, I_y, I_z :

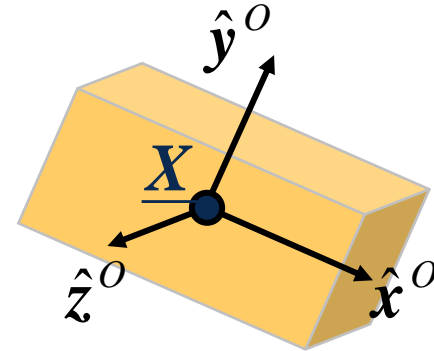


$$I = \int \rho(\vec{r})(\vec{r}^2 - \vec{r}\vec{r}^T) d\vec{r} = \int_{-c/2}^{c/2} \int_{-b/2}^{b/2} \int_{-a/2}^{a/2} \rho_0 \begin{pmatrix} y^2 + z^2 & -xy & -xz \\ -xy & x^2 + z^2 & -yz \\ -xz & -yz & x^2 + y^2 \end{pmatrix} dx dy dz$$

$$= \rho_0 \begin{pmatrix} abc \frac{b^2 + c^2}{12} & 0 & 0 \\ 0 & abc \frac{a^2 + c^2}{12} & 0 \\ 0 & 0 & abc \frac{a^2 + b^2}{12} \end{pmatrix} = \frac{M_0}{12} \begin{pmatrix} b^2 + c^2 & 0 & 0 \\ 0 & a^2 + c^2 & 0 \\ 0 & 0 & a^2 + b^2 \end{pmatrix} = \begin{pmatrix} I_x & 0 & 0 \\ 0 & I_y & 0 \\ 0 & 0 & I_z \end{pmatrix}$$

check out: <http://www.cs.berkeley.edu/~jfc/mirtich/massProps.html>

- Since the tensor of inertia is symmetric and positive definite, one can always find an orthonormal **coordinate system** in which the **tensor is diagonal** and completely defined by I_x , I_y , I_z .
- This is typically used as a **natural coordinate system**.
- **Caution** in contrast to mass, the **inertia tensor must be transformed from the object to the world coordinate system**.
The rotation matrix of the orientation will be multiplied from the left and transposed from the right.



$$\mathbf{I}_O = \begin{pmatrix} I_x & 0 & 0 \\ 0 & I_y & 0 \\ 0 & 0 & I_z \end{pmatrix}$$

$$\mathbf{I}_W = \mathbf{R}\mathbf{I}_O\mathbf{R}^T$$



Dynamics – Angular Momentum

- To the relation between **force and linear momentum** corresponds a similar relation between **torque and angular momentum**
- **Angular momentum \vec{L}** is a **vectorial conserved quantity**
- Inertia tensor changes over time with rotation matrix

$$\mathbf{I}(t) = \mathbf{R}(t)\mathbf{I}_o\mathbf{R}^T(t) \quad \dot{\mathbf{R}} = \boldsymbol{\omega}^* \mathbf{R}$$

$$\dot{\mathbf{I}} = \dot{\mathbf{R}}\mathbf{I}_o\mathbf{R}^T + \mathbf{R}\mathbf{I}_o\dot{\mathbf{R}}^T$$

$$\dot{\mathbf{I}} = \boldsymbol{\omega}^* \mathbf{R}\mathbf{I}_o\mathbf{R}^T - \mathbf{R}\mathbf{I}_o\mathbf{R}^T \boldsymbol{\omega}^* = \boldsymbol{\omega}^* \mathbf{I} - \mathbf{I} \boldsymbol{\omega}^*$$

- \vec{L} and $\vec{\omega}$ are parallel if $\vec{\omega}$ points along a main axis of the inertia tensor, and it holds $\vec{T} = \dot{\vec{L}} = \mathbf{I}\dot{\vec{\omega}}$
- In case of equal momentums of inertia all their linear combinations yield such main axes.

$$\vec{P} = M\vec{V} \Rightarrow \vec{F} = \dot{\vec{P}}$$

$$\Rightarrow \vec{F} = M\dot{\vec{V}}$$

$$\vec{L} = \mathbf{I}\vec{\omega} \Rightarrow \vec{T} = \dot{\vec{L}}$$

$$\Rightarrow \vec{T} = \mathbf{I}\dot{\vec{\omega}} + \dot{\mathbf{I}}\vec{\omega}$$

$$\Rightarrow \vec{T} = \mathbf{I}\dot{\vec{\omega}} + \boldsymbol{\omega}^* \mathbf{I}\vec{\omega} - \underbrace{\mathbf{I} \boldsymbol{\omega}^* \vec{\omega}}_{\vec{0}}$$

$$\Rightarrow \vec{T} = \mathbf{I}\dot{\vec{\omega}} + \boldsymbol{\omega}^* \mathbf{I}\vec{\omega}$$

$$\Rightarrow \vec{T} = \mathbf{I}\dot{\vec{\omega}} + \boldsymbol{\omega}^* \vec{L}$$

Example: nutation of spinning top

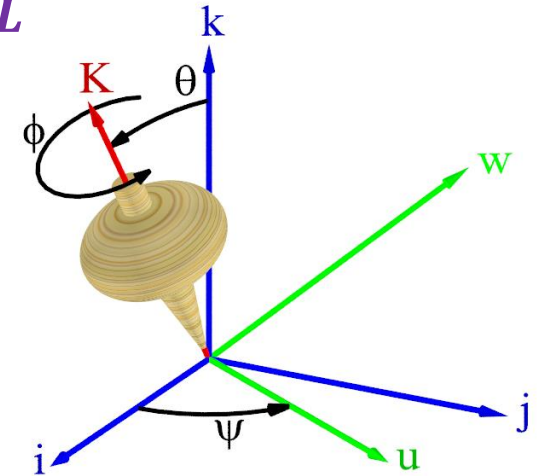
- If top spinning around main axis \vec{K} that is tilted by angle θ from the vertical axis, gravity generates a torque \vec{T} pointing along \hat{u} axis in figure on right side
- This torque rotates main axis \vec{K} around vertical axis
- Angular velocity of precession fulfills

$$\vec{T} = \vec{\omega}_p \times \vec{L}$$

such that $\omega_p = \frac{T}{\sin \theta \cdot L}$

- Precession speeds up with decrease in angular momentum
- If θ changes over time the motion is called nutation which can be described by cones

$$\vec{T} = I\dot{\vec{\omega}} + \vec{\omega} * \vec{L}$$

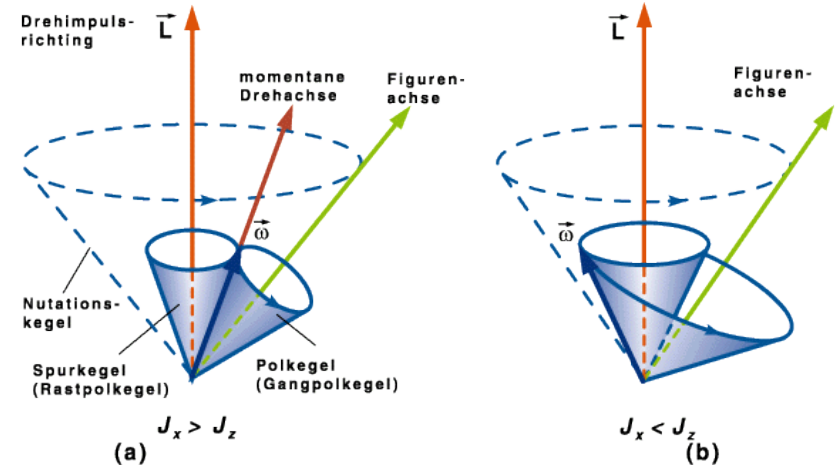


See also: <https://www.youtube.com/watch?v=DG3TuMy0UAM>

Prolate top



Oblate top



Equations of Motion

$$\dot{\mathbf{R}} = \vec{\omega}^* \mathbf{R} \quad \vec{\mathbf{L}} = \mathbf{I} \vec{\omega}$$



- The **state** of the rigid body is uniquely defined by position, orientation, linear and angular momentum.
- The **time evolution function** is computed from previous observations in world space
- Here, only **reciprocal** values of mass and inertia tensor are needed.
- **transformation of the inverse inertia tensor** to world space:

$$\mathbf{I}_W^{-1} = \mathbf{R} \mathbf{I}_O^{-1} \mathbf{R}^T$$

- if a **0** is stored in reciprocal mass or tensor of inertia, this corresponds to an **infinite mass**

$$\vec{\mathbf{y}} = \begin{pmatrix} \underline{\mathbf{X}} \\ \mathbf{R} \\ \vec{\mathbf{P}} \\ \vec{\mathbf{L}} \end{pmatrix}$$

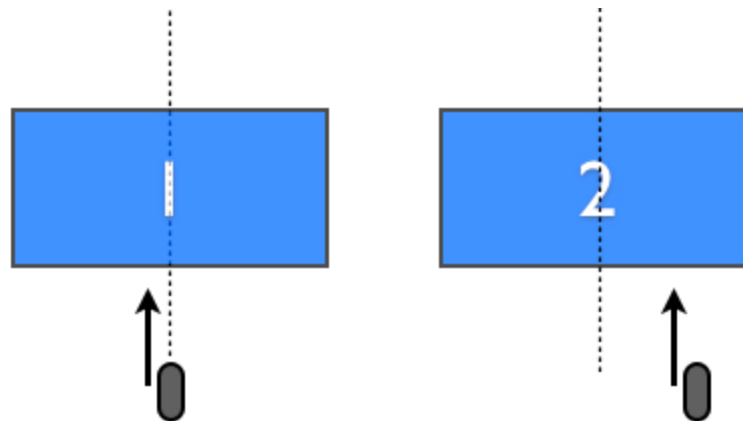
$$\vec{\mathbf{f}}(t, \vec{\mathbf{y}}) = \begin{pmatrix} \underline{\dot{\mathbf{X}}} \\ \dot{\mathbf{R}} \\ \dot{\vec{\mathbf{P}}} \\ \dot{\vec{\mathbf{L}}} \end{pmatrix} = \begin{pmatrix} \frac{1}{M} \vec{\mathbf{P}} \\ \mathbf{I}^{-1} \vec{\mathbf{L}}^* \mathbf{R} \\ \vec{\mathbf{F}}_{ges}(t, \vec{\mathbf{y}}) \\ \vec{\mathbf{T}}_{ges}(t, \vec{\mathbf{y}}) \end{pmatrix}$$

- **Caution:** \mathbf{R} must be **orthogonalized** after each integration step. This can be done, for example, with **polar decomposition**.



Bullet Block Experiment

- <https://www.youtube.com/watch?v=vWVZ6APXM4w>



- Which block flies higher?